

## SDND106 – Shoe rubbing with coefficients of static and dynamic friction

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### Summary:

The purpose of this CAS-test is to validate the functionality of nonlinearity of friction by penalization, with the use of two coefficients of friction, a statics (for the phases of adherence) and a dynamics (for the phases of slip).

The studied system contains two degrees of freedom. Its evolution contains a transition adherence – slip and a transition slip – adherence.

The reference solution is analytical.

## 1 Problem of reference

### 1.1 Geometry

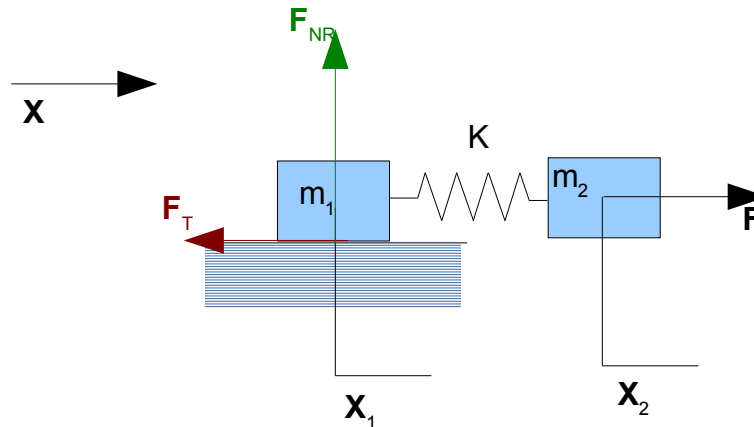


Figure 1.1-a : Diagram of the CAS-test.

### 1.2 Properties of material

The two masses are identical and are worth  $m_1 = m_2 = m = 5 \text{ kg}$ . The stiffness placed between these masses is of  $k = 10^4 \text{ N/m}$ .

### 1.3 Boundary conditions and loadings

Masses  $m_1$  and  $m_2$  move according to  $\vec{x}$  only.

The module of the normal force  $F_N$  is constant equal to  $F_N = 10^4 \text{ N}$ .

Mass  $m_1$  of a condition of contact – friction is affected. Coefficients of friction 5 are:

- in the static case:  $\mu_S = 0,3$ ,
- in the dynamic case:  $\mu_D = 0,2$ .

In this modeling, the system passes by three phases:

- A phase of adherence, during which the force exerted on the mass  $m_2$  is constant:  $F = 3 \cdot 10^3 \text{ N}$ .
- A phase of slip, beginner with  $t_1$ , when the tangential force cannot compensate for the force of traction exerted any more by the spring on  $m_1$ . At the end of an arbitrary time ( $t_2 = 0,2 \text{ s}$ ), the imposed force is put at 0 and the kinetics of the system tends then towards 0.
- One second phase of adherence, beginner with  $t_3$ , when the speed of the mass  $m_1$  cancel yourself.

### 1.4 Initial conditions

At the moment  $t=0$ , the two masses are at rest (displacement and a speed worthless).

## 2 Reference solution

### 2.1 Method of calculating

The pulsation of the system is noted  $\omega_0$  and is such as  $\omega_0^2 = \frac{k}{m}$ .

#### Phase of adherence

At the moment  $t=0$ , mass  $m_1$  check

$$x_1(t)=0$$

$$\dot{x}_1(t)=0$$

and masses it  $m_2$  check

$$x_2(t)=0$$

$$\dot{x}_2(t)=0.$$

The basic principle of dynamics applied to the mass  $m_2$  allows to write the following equation

$$\ddot{x}_2 + \omega_0^2 x_2 = \frac{F}{m},$$

and on the mass  $m_1$  the following equation

$$F_T = -k x_2$$

The general solution of  $x_2$  is form:

$$x_2 = \tilde{A} \cos(\omega_0 t) + \tilde{B} \sin(\omega_0 t) + \frac{F}{(m \omega_0^2)}$$

where  $\tilde{A}$  and  $\tilde{B}$  are constants.

By taking account of the initial conditions, the displacement of the mass  $m_2$  is thus written

$$x_2 = \frac{F}{k} [1 - \cos(\omega_0 t)].$$

This expression is valid until

$$\|F_T\| = \mu F_N$$

In other words, until the moment  $t_1$  checking the following expression

$$F [1 - \cos(\omega_0 t_1)] = \mu_S F_N$$

The phase of slip thus starts with  $t_1$  defined by

$$t_1 = \frac{1}{\omega_0} \arccos\left(1 - \mu_S \frac{F_N}{F}\right)$$

#### Phase of slip

At the moment  $t=t_1$ , mass  $m_1$  check

$$x_1(t_1)=0$$

$$\dot{x}_1(t_1)=0$$

The basic principle of dynamics applied to the mass  $m_1$  allows to write the equation

$$\ddot{x}_1 + \omega_0^2 x_1 = \frac{-\mu_D F_N}{m} + \omega_0^2 x_2,$$

and on the mass  $m_2$  the equation

$$\ddot{x}_2 + \omega_0^2 x_2 = \frac{F}{m} + \omega_0^2 x_1.$$

By making the change of variables

$$\begin{cases} X = x_1 + x_2 \\ Y = x_1 - x_2 \\ \Omega_0 = \sqrt{2} \omega_0 \end{cases}$$

the preceding system is written

$$\begin{aligned} \ddot{X} &= \frac{F - \mu_D F_N}{m} \\ \ddot{Y} + \Omega_0^2 Y &= -\frac{F + \mu_D F_N}{m} \end{aligned}$$

For  $t \geq t_1$ , the general solution of  $X$  is form

$$X = \frac{(F - \mu_D F_N)}{2m} (t - t_1)^2 + \tilde{C} (t - t_1) + \tilde{D}.$$

and that of  $Y$  is form

$$Y = \tilde{E} \cos(\Omega_0(t - t_1)) + \tilde{F} \sin(\Omega_0(t - t_1)) - \frac{F + \mu_D F_N}{2k}.$$

By taking account of the initial conditions, expressions of the constants  $\tilde{C}$ ,  $\tilde{D}$ ,  $\tilde{E}$  and  $\tilde{F}$  are worth:

$$\begin{aligned} \tilde{C} &= \dot{x}_2(t_1) \\ \tilde{D} &= x_2(t_1) \\ \tilde{E} &= \frac{F + \mu_D F_N}{2k} - x_2(t_1) \\ \tilde{F} &= -\frac{\dot{x}_2(t_1)}{\Omega_0} \end{aligned}$$

Displacements  $x_1$  and  $x_2$  are deduced from the preceding expressions.

**Phase of slip with a loading  $F=0$  from  $t_2$  (arbitrary but higher than  $t_1$ ) until the moment  $t_3$  (moment of return to the phase adhesion)**

The basic principle of dynamics presented previously is always valid. For the mass  $m_1$ , the checked equation is the same one:

$$\ddot{x}_1 + \omega_0^2 x_1 = \frac{-(\mu_D F_N)}{m} + \omega_0^2 x_2,$$

and for the mass  $m_2$

$$\ddot{x}_2 + \omega_0^2 x_2 = \omega_0^2 x_1$$

With the same change of variable as previously, the system is written

$$\begin{aligned} \ddot{X} &= -\frac{\mu_D F_N}{m} \\ \ddot{Y} + \Omega_0^2 Y &= -\frac{\mu_D F_N}{m} \end{aligned}$$

For  $t > t_2$ , general solution of  $X$  is form

$$X = -\frac{\mu_D F_N}{2m} (t - t_2)^2 + \tilde{G} (t - t_2) + \tilde{H}$$

The general solution of  $Y$  is form

$$Y = \tilde{I} \cos(\Omega_0(t - t_2)) + \tilde{J} \sin(\Omega_0(t - t_2)) - \frac{\mu_D F_N}{2k}$$

By taking account of the initial conditions, expressions of the constants  $\tilde{G}$ ,  $\tilde{H}$ ,  $\tilde{I}$  and  $\tilde{J}$  are worth:

$$\tilde{G} = \dot{x}_1(t_2) + \dot{x}_2(t_2)$$

$$\tilde{H} = x_1(t_2) + x_2(t_2)$$

$$\tilde{J} = \frac{\mu_D F_N}{2k} + x_1(t_2) - x_2(t_2)$$

$$\tilde{I} = \frac{\dot{x}_1(t_2) - \dot{x}_2(t_2)}{\Omega_0}$$

Displacements  $x_1$  and  $x_2$  are deduced from the preceding expressions.

## 2.2 Sizes and results of reference

The sizes tested are the kinematics of the masses  $m_1$  and  $m_2$  at various moments in the various phases of modeling. Are also tested the following moments of transition:

- passage de la phase of adherence to the phase of slip (urgent  $t_1$ );
- passage de la phase of slip to the phase of adherence (moment  $t_3$ ).

## 2.3 Uncertainties on the solution

Exact analytical solution.

## 2.4 Bibliographical references

[1] E. BOYERE: Modeling of the shocks and friction in transitory analysis by modal recombination. Reference material of Code\_Aster R5.06.03. September 2009.

## 3 Modeling A

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### 3.1 Characteristics of modeling

In this modeling, the force exerted on the mass  $m_2$  cancel yourself in  $t_2=0,2 s$ .

### 3.2 Characteristics of the grid

The grid contains 2 discrete meshes to a node (for each mass) and 1 discrete mesh for the spring.

### 3.3 Sizes tested and results

Type	Moment ( s )	Size	Reference	Aster	Difference (%)
Analytical	0.02	$x_1$	0.	2.0621E-05	
Analytical	0.02	$x_2$	0.11221	0.11221	0
Analytical	0.15	$x_1$	1.35332	1.35329	-0,002
Analytical	0.15	$x_2$	1.80751	1.80751	0
Analytical	0.34	$\dot{x}_1$	0.	-3.3802E-5	
Analytical	0.34	$x_2$	3.96813	3.97012	0.05

Type	Size (S)	Reference	Aster	% difference
Analytical	$t_1$	0.03512	0.03520	0.23
Analytical	$t_3$	0.31492	0.31500	0.03

### 3.4 Remarks

The difference in % is not given for the cases where the value of reference is worth 0, but it is observed that the absolute difference is about  $10^{-5}$ .

## 4 Summary of the results

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This CAS-test shows that the transitions adherence – slip and slip – adherence are well collected. It validates the capacity of the operator in addition `DYNA_VIBRA` to integrate problems in great displacements in translation.