

## SSNL107 - Embedded plate subjected to an inflection by beams in contact with the free edge

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### Summary:

This test validates the unilateral contact between elements of beam `POU_D_E` (right beam of Euler) and of the elements of hull `DKQ`.

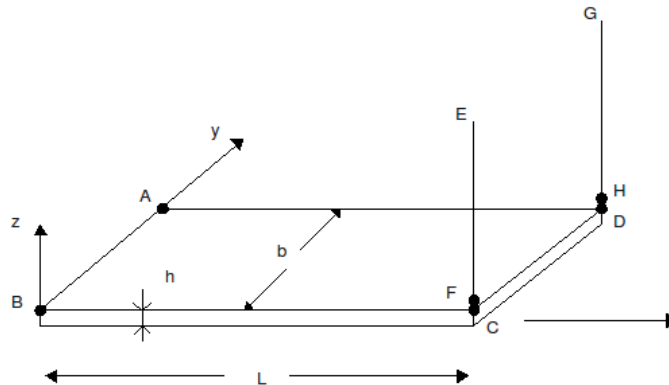
The main features are:

- linear behavior,
- elastic analysis,
- unilateral contact,
- 2 modelings: elements `POU_D_E` and `DKQ` while using `CONTACT` in `AFFE_CHAR_MECA` and in `AFFE_CHAR_MECA_F`.

The reference solution is analytical and the got results are of good quality.

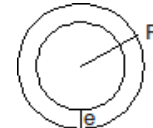
## 1 Problem of reference

### 1.1 Geometry



|            |              |                    |
|------------|--------------|--------------------|
| Plate ABCD | of length    | $L = 10\text{mm}$  |
|            | of width     | $B = 1\text{mm}$   |
|            | of thickness | $H = 0.1\text{mm}$ |

|                   |                      |                                   |
|-------------------|----------------------|-----------------------------------|
| 2 beams EF and GH | Of length            | $l = 1\text{mm}$                  |
|                   | Of section circulars | $R = 2.10^{-3}$ , $e = 2.10^{-4}$ |



### 1.2 Material properties

Linear elasticity:  $E = 210^5 \text{MPa}$ ,  $\nu = 0.3$   
Identical for the plate and the two beams.

### 1.3 Boundary conditions and loadings

Embedding on  $AB$ :  $DX = DY = DZ = DRX = DRY = DRZ = 0$   
Displacement imposed in  $E$  and  $G$ :  $DZ = -0.2\text{mm}$   
Unilateral contact enters  $F$  and  $C$  and enters  $H$  and  $D$

### 1.4 Initial conditions

Without object.

## 2 Reference solution

### 2.1 Method of calculating used for the reference solution

Analytical

The plate undergoes a pure bending. The solution is of the type "poutre" :

$$V = DZ(C) = DZ(D) = \frac{PL^3}{3E.I_y} \quad \text{with } I_y = \frac{bh^3}{12}$$

The arrow  $V$  and charges it  $P$  are unknown.

The two beams are in pure compression:

$$-P = 2 \cdot \frac{ES}{L} (V - U) \quad \text{with } U = DZ(E) \\ = DZ(G)$$

One can thus find  $P$  and  $V$  starting from these two equations. One obtains:

$$P = \frac{6E S I U}{2SL^3 + 3I_y l}$$

$$V = \frac{2SL^3 U}{2SL^3 + 3I_y l}$$

### 2.2 Results of reference

$$V = -0.19005 \text{ mm}$$

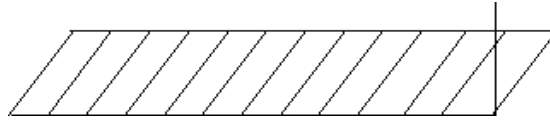
$$P = -9.5025 \cdot 10^{-3} \text{ N}$$

### 2.3 Uncertainty on the solution

Worthless. Analytical solution.

## 3 Modeling B

### 3.1 Characteristics of modeling



20 elements of hull DKQ  
2 elements of beam POU\_D\_E

There exists a game (  $0.2\text{ mm}$  ) between the points  $H$  and  $D$  in the grid.

One introduces a fictitious game (  $0.2\text{ mm}$  ) between the points  $F$  and  $C$  by the keyword `DIST_ESCL` of `DEFI_CONTACT` with `FORMULATION=' DISCRETE'`.

The contact is treated between the meshes `POI1` thanks to the keyword `ESCL_FIXE` of `DEFI_CONTACT`.

### 3.2 Characteristics of the grid

Many nodes: 46

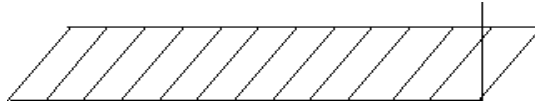
Many meshes and types: 20 QUAD4, 2 SEG2

### 3.3 Sizes tested and results

| Identification |      |       | Reference                | % tolerance |
|----------------|------|-------|--------------------------|-------------|
| $C$            | $DZ$ | $N46$ | $-0.19005$               | 0.03        |
| $D$            | $DZ$ | $N45$ | $-0.19005$               | 0.03        |
| $EF$           | $N$  | $M22$ | $-4.75126 \cdot 10^{-3}$ | 0.58        |
| $GH$           | $N$  | $M21$ | $-4.75126 \cdot 10^{-3}$ | 0.58        |

## 4 Modeling C

### 4.1 Characteristics of modeling



20 elements of hull DKQ  
2 elements of beam POU\_D\_E

There exists a game (  $0.2\text{ mm}$  ) between the points  $H$  and  $D$  in the grid.

One introduces a fictitious game between the points  $F$  and  $C$  by the keyword `DIST_ESCL` of `DEFI_CONTACT`. This game is declared like a function of time, of constant value equalizes with  $0.2\text{ mm}$  .

This problem is solved in `FORMULATION=' DISCRETE '` .

### 4.2 Characteristics of the grid

Many nodes: 46

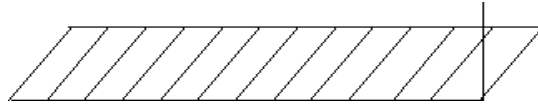
Many meshes and types: 20 QUAD4, 2 SEG2

### 4.3 Sizes tested and results

| Identification |      |       | Reference                 | % tolerance |
|----------------|------|-------|---------------------------|-------------|
| $C$            | $DZ$ | $N46$ | $- 0.19005$               | 0.10        |
| $D$            | $DZ$ | $N45$ | $- 0.19005$               | 0.10        |
| $EF$           | $N$  | $M22$ | $- 4.75126 \cdot 10^{-3}$ | 1.00        |
| $GH$           | $N$  | $M21$ | $- 4.75126 \cdot 10^{-3}$ | 1.00        |

## 5 Modeling D

### 5.1 Characteristics of modeling



20 elements of hull DKQ  
2 elements of beam POU\_D\_E

There exists a game (  $0.2\text{ mm}$  ) between the points  $H$  and  $D$  in the grid.  
One introduces a fictitious game between the points  $F$  and  $C$  by the keyword `DIST_ESCL` of `DEFI_CONTACT`. This game is declared like a constant equalizes with  $0.2\text{ mm}$  .  
This problem is solved by the algorithm `GCP` in `FORMULATION=' DISCRETE '` .

### 5.2 Characteristics of the grid

Many nodes: 46  
Many meshes and types: 20 QUAD4, 2 SEG2

### 5.3 Sizes tested and results

| Identification |      |       | Reference                | % tolerance |
|----------------|------|-------|--------------------------|-------------|
| $C$            | $DZ$ | $N46$ | $-0.19005$               | 0.10        |
| $D$            | $DZ$ | $N45$ | $-0.19005$               | 0.10        |
| $EF$           | $N$  | $M22$ | $-4.75126 \cdot 10^{-3}$ | 1.00        |
| $GH$           | $N$  | $M21$ | $-4.75126 \cdot 10^{-3}$ | 1.00        |

## 6 Summary of the results

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The results are very close to the analytical solution ( 0.58% ). They are not exact because they depend on the smoothness of the grid of the plate.

The results show the good performance of the unilateral contact between the beams and the plate.