

## PERF001 - Elastic design of a hollow ring subjected to an internal pressure

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### Summary:

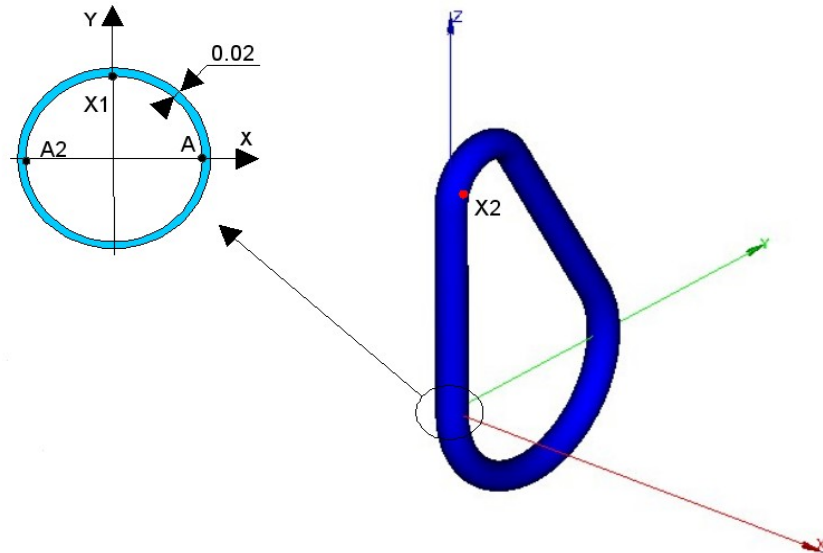
The objective of this CAS-test is to measure the performances of an elastic design 3D.

Five modelings carried out are the following ones:

- Modeling a: grid HEXA8,  $5.0 E5$  degrees of freedom, MECA\_STATIQUE ( 'MULT\_FRONT' )
- Modeling b: grid HEXA20,  $5.2 E5$  degrees of freedom, MECA\_STATIQUE ( 'MULT\_FRONT' )
- Modeling C: grid HEXA8,  $1.0 E6$  degrees of freedom, MECA\_STATIQUE ( 'MULT\_FRONT' )
- Modeling D: grid HEXA8,  $2.0 E6$  degrees of freedom, MECA\_STATIQUE ( 'MULT\_FRONT' )
- Modeling E: grid HEXA8,  $5.0 E5$  degrees of freedom, MECA\_STATIQUE ( 'MUMPS' )

## 1 Problem of reference

### 1.1 Geometry



Coordinates of the points (m) :

$A : (1., 0., 0.)$   
 $A2 : (-1., 0., 0.)$   
 $X1 : (0., 1., 0.)$   
 $X2 : (1., 0., 15.)$

Group of meshes: *PI* internal surface

### 1.2 Properties of material

- $E = 5.0 E11 Pa$
- $\nu = 0.3$
- $\rho = 9800 kg.m^{-3}$

### 1.3 Boundary conditions and loadings

- Imposed displacements:
  - $A$  :  $DX = DY = DZ = 0.$
  - $A2$  :  $DY = DZ = 0.$
  - $X1$  :  $DZ = 0.$
- Internal pressure:
  - $p = 2.0 E6 Pa$

## 2 Reference solution

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### 2.1 Method of calculating

The result of reference (displacement following the axis  $Z$  point  $X2$ ) was obtained by making the average of the displacements calculated at the time of modelings  $A$ ,  $B$  and  $C$ .

### 2.2 Results of reference

Displacement at the point  $X2$  :  $DZ = 5.87E - 4 m$

### 2.3 Uncertainties

Digital solution.

## 3 Modeling A

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### 3.1 Characteristics of modeling A

Modeling 3D:

Many nodes	168 000	
Many meshes	225 248	That is to say:
		SEG2 6 128
		QUAD4 93 120
		HEXA8 126000

### 3.2 Results

Points	Size	Reference ( <i>m</i> )	Tolerance (%)
<i>X2</i>	<i>DZ</i>	5.870E-4	3.000E-3

## 4 Modeling B

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### 4.1 Characteristics of modeling B

Modeling 3D:

Many nodes	172 800	
Many meshes	62 408	That is to say:
		SEG3 2352
		QUAD8 26 496
		HEXA20 34 560

### 4.2 Results

Points	Size	Reference (m)	Tolerance (%)
<i>X2</i>	<i>DZ</i>	5.870E-4	3.000E-3

## 5 Modeling C

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### 5.1 Characteristics of modeling C

Modeling 3D:

Many nodes	336 000		
Many meshes	405 472	That is to say:	
		SEG2	6 192
		QUAD4	105 280
		HEXA8	294 000

### 5.2 Results

Points	Size	Reference ( <i>m</i> )	Tolerance (%)
<i>X2</i>	<i>DZ</i>	5.870E-4	3.000E-3

## 6 Modeling D

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### 6.1 Characteristics of modeling D

Modeling 3D:

Many nodes	672000		
Many meshes	785 632	That is to say:	
		SEG2	6 672
		QUAD4	190 960
		HEXA8	588 000

### 6.2 Results

Points	Size	Reference ( <i>m</i> )	Tolerance (%)
<i>X2</i>	<i>DZ</i>	5.870E-4	3.000E-3

## 7 Modeling E

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### 7.1 Characteristics of modeling E

Modeling 3D:

Many nodes	168000		
Many meshes	225248	That is to say:	
		SEG2	6128
		QUAD4	93120
		HEXA8	126000

### 7.2 Results

Points	Size	Reference ( <i>m</i> )	Tolerance (%)
<i>X2</i>	<i>DZ</i>	5.870E-4	3.000E-3



## 8 Summary of the results

Machine	Aster	MOD.	Nb DDL	Memory (Mo)		Time execution ( MECA_STATIQUE ) (dryness)			
				Allocat ed	Used	USERS	SYSTEM	USERS+SYS	ELAPSED
Linux 64 bits (ia64) "Bull"	10.1	With	504,012	763	758	138.22	18.73	156.37	157.49
		B	518,412	1,348	1,221	406.03	50.93	456.96	462.52
		C	1 008,012	1,690	1,263	707.14	97.74	804.88	807.53
		D	2 016,012	3,961	2,486	2256.55	225.42	2481.97	2633.52
		E	504,012	624	570	132.98	35.57	168.55	169.16