

## SSLV306 - Beam 3D in imposed displacements

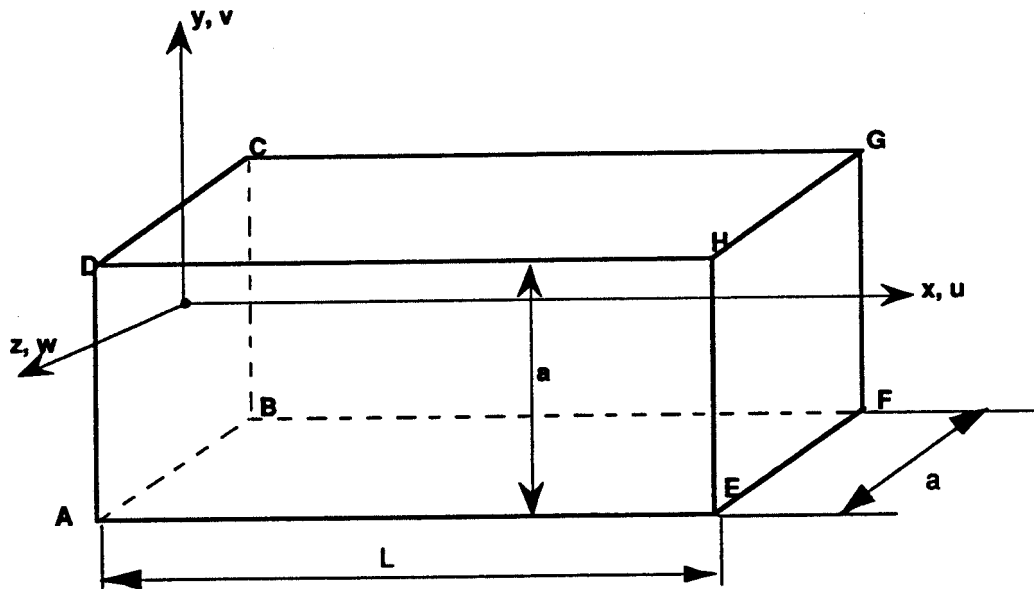
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### Summary:

The purpose of the test is to validate the displacements imposed on faces (`FACE_IMPO`), their values being variable in space. These values are imposed at the end of a beam 3D, modelling a bending strain.

## 1 Problem of reference

### 1.1 Geometry



Length :  $L=2\text{ m}$   
 Square section, on side :  $a=0.2\text{ m}$   
 Moment of inertia :  $I=1.333\times 10^{-4}\text{ m}^4$

### 1.2 Material properties

$E=2.1\times 10^{11}\text{ Pa}$   
 $\nu=0.3$

### 1.3 Boundary conditions and loadings

Embedding of the section  $ABCD$

Displacement imposed on the face  $EFGH$  :

- constant  $v_o$  in the direction  $y$ ,  $v_o=0.952\times 10^{-5}\text{ m}$
- varying according to the position  $y$  point of the section, and being worth:  
 $u_o=-y\theta_o$ ,  $\theta_o=0.714\times 10^{-5}\text{ radians}$

### 1.4 Initial conditions

Without object for the static analysis.

## 2 Reference solution

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### 2.1 Method of calculating used for the reference solution

The displacements imposed equivalent on a force applied at the end of resultant:

$$F = \frac{3EI}{L^3} v_o = 100N$$

$\theta_o$  represent the rotation of the section  $EFGH$ :

$$\theta_o = \frac{FL^2}{2EI}$$

The bending stress  $\sigma_{xx}$  with embedding is worth then:

$$\sigma_{xx}(ABCD) = \pm \frac{FL}{I/y}$$

### 2.2 Results of reference

- 1) Displacement  $v$  points  $E, F, G, H$
- 2) Bending stresses  $\sigma_{xx}$  at the points  $A, B, C, D$

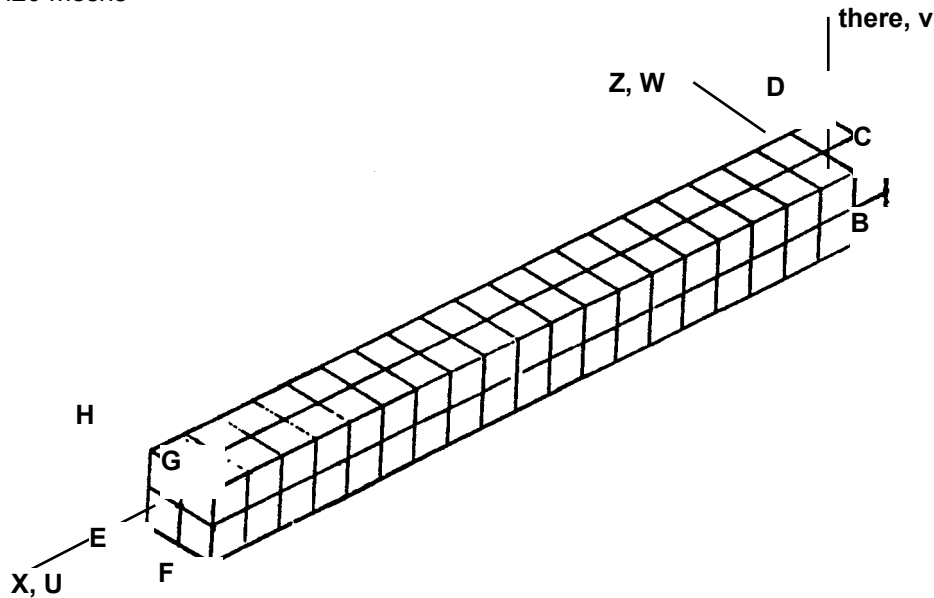
### 2.3 Uncertainty on the solution

Analytical solution.

## 3 Modeling A

### 3.1 Characteristics of modeling

3D, H20 meshes



Loading by displacements imposed on face  $EFGH$  :

$$\begin{aligned} DY &: 0.952 \times 10^{-5} \\ DX &: \text{function of } y \text{ defined in 2 points:} \end{aligned} \quad \begin{aligned} f(0) &= 0 \\ f(0,1) &= -0.0714E-5 \end{aligned}$$

Cutting:

- 1) 20 elements according to the length
- 2) 2 elements according to the width and the thickness

### 3.2 Characteristics of the grid

Many nodes: 621  
Many meshes and types: 80 HEXA20

## 4 Results of modeling A

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### 4.1 Values tested

| Localization        | Type of value     | Reference              | Aster                  | % difference |
|---------------------|-------------------|------------------------|------------------------|--------------|
| Points $E, F, G, H$ | $v(m)$            | $9.52 \times 10^{-6}$  | $9.52 \times 10^{-6}$  | 0            |
| Points $E, F$       | $u(m)$            | $7.14 \times 10^{-7}$  | $7.14 \times 10^{-7}$  | 0.           |
| Points $G, H$       | $u(m)$            | $-7.14 \times 10^{-7}$ | $-7.14 \times 10^{-7}$ | 0.           |
| Points $A, B$       | $\sigma_{xx}(Pa)$ | $1.5 \times 10^5$      | $1.64 \times 10^5$     | 9.5          |
| Points $C, D$       | $\sigma_{xx}(Pa)$ | $-1.5 \times 10^5$     | $-1.64 \times 10^5$    | -9.5         |

## 5 Summary of the results

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The functionality "displacements imposed function" provides the expected results; the values of bending stresses are satisfactory, given that with the dealt problem is a problem of inflection.