

SSNA302 - Simply supported circular plate subjected to pressure

Summary:

This test consists in applying a transverse pressure to a circular plate, simply supported and made up of an elastic material. It is intended particularly to study the taking into account of nonthe geometrical linearities in the absence of initial curve when the transverse stiffness is only due to the effect of plate.

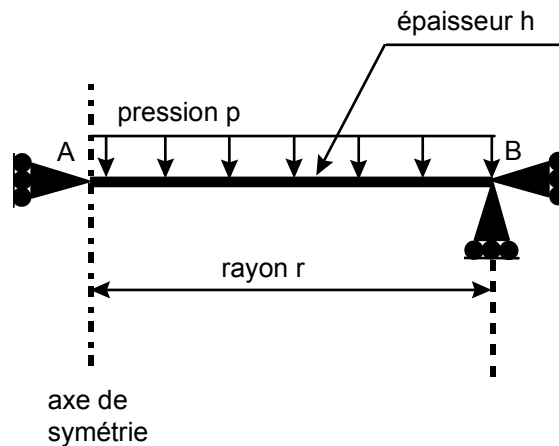
Modeling is made with voluminal elements of type HEXA20 and PENTA15 and surface elements of type QUAD8 and TRIA6 for the application of the pressure.

The reference is software the SAMCEF software. One has for information the results resulting from the theory of the thin hulls.

Under the keyword `BEHAVIOR`, the option `GREEN` is compared with the option `PETIT_REAC`.

1 Problem of reference

1.1 Geometry



1.2 Material properties

Isotropic elastic material:

$$E = 200000 \text{ Mpa}$$
$$\nu = 0.3$$

1.3 Boundary conditions and loadings

Not B :

$$u_x = 0$$
$$u_y = 0.$$

A transverse pressure is applied p on the plate: $p = 222.72 \text{ N/mm}^2$. This pressure corresponds to an arrow w_0 of 1.5 mm .

Ray $r = 10 \text{ mm}$
Thickness $h = 1 \text{ mm}$

The problem is axisymmetric.

2 Reference solution

2.1 Reference solution

The reference is software the SAMCEF software. One has, for information, in the paragraph [§2.2], the theoretical results related to an assumption of type thin hull. Then, the results got with the SAMCEF software are presented according to whether one chooses an assumption of type thick hull or standard volume. It is the latter which is taken into account for the evaluation of *Code_Aster*.

2.2 Analytical solution and results of reference

The following formula gives the arrow w_0 in the center of the plate:

$$\frac{w_0}{h} + A \left(\frac{w_0}{h} \right)^3 = \frac{Bp}{E} \left(\frac{r}{h} \right)^4 \quad \text{with } A=1.852 \quad \text{and } B=0.696$$

The constraints with mid thickness are worth:

$$\sigma_{rr} = \alpha_r E \frac{w_0}{r^2}$$

$$\sigma_{\theta\theta} = \alpha_t E \frac{w_0}{r^2}$$

The constraints in lower skin are worth

$$\sigma_{rr}' = \beta_r E \frac{w_0 h}{r^2}$$

$$\sigma_{\theta\theta}' = \beta_t E \frac{w_0 h}{r^2}$$

The coefficients are worth:

In the center of the plate:

$$\alpha_r = \alpha_t = 0.905$$

$$\beta_r = \beta_t = 1.778$$

At the edge of the plate:

$$\alpha_r = 0.610$$

$$\alpha_t = 0.183$$

$$\beta_r = 0$$

$$\beta_t = 0.755$$

For a pressure of 222.72 MPa , the arrow w_0 is worth 1.5 mm and the following constraints are obtained:

Position	σ_{rr} (MPa)	$\sigma_{\theta\theta}$ (MPa)	σ_{rr}' (MPa)	$\sigma_{\theta\theta}'$ (MPa)
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	4072.5	4072.5	5334.0	5334.0
Center				

These results correspond to an assumption of thin hull.

The following table shows the results got by the SAMCEF software for a modeling of type thick hull and standard volume.

Identification	Thick hull	Volume
Arrow w_0 (mm)	- 1.43041 E-3	- 1.441838 E-3
SIXX (MPa) center, mid thickness	3899.88	3850.88
SIYY (MPa) center, mid thickness	3899.53	3850.91
SIXX (MPa) center, skin inf	8085.81	8133.60
SIYY (MPa) center, skin inf	8083.32	8133.65
SIXX (MPa) $r = R/2$, mid thickness	3596.91	3512.79
SIYY (MPa) $r = R/2$, mid thickness	3056.37	2947.55
SIXX (MPa) $r = R/2$, skin inf	7798.18	7815.73
SIYY (MPa) $r = R/2$, skin inf	7264.69	7307.04

The values of the constraints are values by element extrapolated with the nodes.

The results got with the SAMCEF software are close one to the other.

The variations of results between the mean theory hull and calculation finite element using the assumption thick hull are important.

One chooses to take as reference the voluminal calculation obtained by the SAMCEF software.

2.3 Bibliographical references

- 1) Theory of Plates and Shells, Timoshenko S.P., 2nd edition, p 412

3 Modeling A

3.1 Characteristics of modeling A

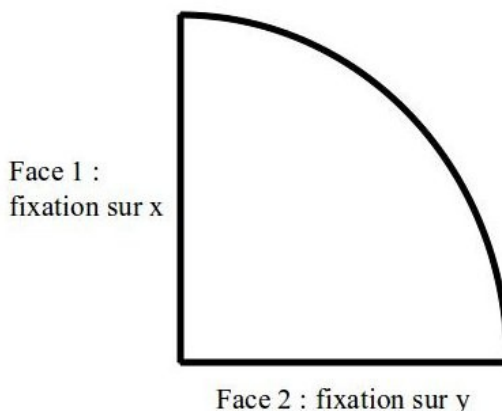


Figure 3.1-3.1-a

Only one quarter of plate is modelled. One introduces conditions of symmetry on the two faces shown above.

Moreover one fixes according to x, y, z all nodes of the edge located at semi thickness.

In order to most accurately represent the assumptions of thin hull, one introduces linear constraints on the degrees of freedom of the nodes of the edge. Those C_i are written:

for two nodes i and j located on both sides of the average layer:

$$u_i + u_j = 0$$

$$v_i + v_j = 0$$

where U and v indicate displacements along the axes x and y .

The syntax used in the data file of *Code_Aster* is the following one:

```
LIAISON_DDL (NODE:  $N_i$   $N_j$  ) , DDL: ('DX', 'DX') , COEF_MULT: (1, 1) ,  
COEF_IMPO: 0
```

One applies the pressure linearly by using 6 increments.

3.2 Characteristics of the grid

Many nodes:	2091
Many meshes and types:	368 HEXA20, 28 PENTA15 92 QUAD8, 7 TRIA6

3.3 Values tested

Identification

Reference

Arrow w_0 (mm)		- 1.441838E-3
SIXX (MPa)	center, mid thickness	3850.9
SIYY (MPa)	center, mid thickness	3850.9
SIXX (MPa)	center, skin inf	8133.6
SIYY (MPa)	center, skin inf	8133.6
SIXX (MPa)	$r = R/2$, mid thickness	3512.8
SIYY (MPa)	$r = R/2$, mid thickness	2947.5
SIXX (MPa)	$r = R/2$, skin inf	7815.7
SIYY (MPa)	$r = R/2$, skin inf	7307.0

3.4 Remarks

One obtains the same precision on the results by using only one increment of loading.

4 Modeling B

4.1 Characteristics of modeling B

Idem modeling A, but by treating nonthe geometrical linearities by the keyword BEHAVIOR option PETIT_REAC.

4.2 Characteristics of the grid

Many nodes: 2091
Many meshes and types: 368 HEXA20, 28 PENTA15
92 QUAD8, 7 TRIA6

4.3 Values tested

Identification	Reference	Aster	% difference
Arrow $w0$ (mm)	- 1.441838E-3	- 1.3692E-3	- 5,036
SIXX (MPa) center, mid thickness	3850.9	4219.3	9,568
SIYY (MPa) center, mid thickness	3850.9	4219.4	9,569
SIXX (MPa) center, skin inf	8133.6	7950.5	- 2,251
SIYY (MPa) center, skin inf	8133.6	7950.5	- 2,252
SIXX (MPa) $r = R/2$, mid thickness	3512.8	3838.4	9,268
SIYY (MPa) $r = R/2$, mid thickness	2947.5	3314.2	12,440
SIXX (MPa) $r = R/2$, skin inf	7815.7	7593.9	- 2,839
SIYY (MPa) $r = R/2$, skin inf	7307.0	7123.7	- 2,509

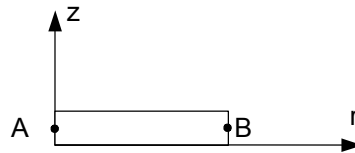
4.4 Remarks

One notes an important difference between the reference solution and the solution provided by Code_Aster.

It is checked that this variation tends towards 0 when one increases the number of increments.

5 Modeling C

5.1 Characteristics of modeling C



- axisymmetric modeling
- boundary conditions:

$$B : \quad DX = 0 \quad DY = 0$$

As for modeling A (3D), of the linear constraints are introduced for better representing a kinematics of hull. They relate to the nodes of the edge external of the disc. If i and j 2 nodes on both sides of the average layer indicate, they are written:

$$u_i + u_j = 0 \text{ where } u \text{ indicate radial displacement}$$

5.2 Characteristics of the grid

Many nodes: 149
Many meshes and types: 40 QUAD8, 10 SEG3

5.3 Values tested

Identification	Reference
Arrow w_0 (mm)	- 1,430
SIXX (MPa) center, mid thickness	3900
SIZZ (MPa) center, mid thickness	3900

5.4 Remarks

In the absence of an axisymmetric calculation of reference the SAMCEF software, one is based on calculation in thick hull (always the SAMCEF software) and one does not compare that the arrow and the constraints with the center with mid thickness.

6 Summary of the results

The performances in time calculation and precision of the results are satisfactory while using the keyword `BEHAVIOR` option `GREEN`. On the other hand, treatment of nonthe geometrical linearities by the keyword `BEHAVIOR` option `PETIT_REAC` provides results rather far away from the reference solution by adopting a discretization in $\bar{6}$ pas de time which leads to costs in already important times calculation.