

## SSLL101 - Piping: Problem of HOVGAARD

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### Summary:

It is about a linear elastic test, in statics, of a noncoplanar three-dimensional piping comprising elbows. There exists a test in dynamics of same structure (SDLX02) [V2.05.002].

The elements are tested `POU_D_T`, `PIPE (SEG3 and SEG4)` and `TUYAU_6M (SEG3)` via 5 modelings:

- modeling a: 92 elements `POU_D_T` (40 for an elbow), calculation with `MECA_STATIQUE`,
- modeling C: 28 elements `PIPE (SEG3)` (5 for an elbow), calculation with `MECA_STATIQUE`,
- modeling D: 28 elements `TUYAU_6M (SEG3)` (5 for an elbow), calculation with `MECA_STATIQUE`,
- modeling E: 28 elements `PIPE (SEG4)` (5 for an elbow), calculation with `MECA_STATIQUE`,

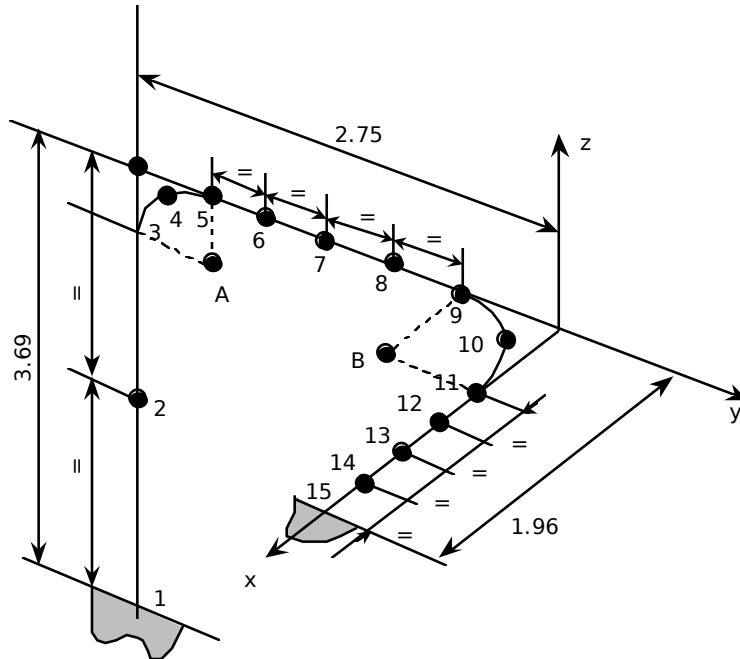
The loadings are of type:

- gravity,
- thermics,
- nodal forces.

## 1 Problem of reference

### 1.1 Geometry

Geometry, as well as the points of modeling are represented on the following figure:



$$A \begin{vmatrix} 0. \\ -1.828 \\ -0.922 \end{vmatrix}$$

$$B \begin{vmatrix} 0.922 \\ -0.922 \\ 0. \end{vmatrix}$$

Lengths in meters

- diameter external of the pipe: 0.185 m
- thickness of the pipe: 6.12 mm
- radius of curvature of the elbows: 0.922 m
- piping full of water

### 1.2 Material properties

$$E = 1.658 E + 11 Pa$$

$$\nu = 0.3$$

$$\rho = 13404.10 kg/m^3$$

$$\alpha = 0.1288 E - 4 / C^\circ$$

### 1.3 Boundary conditions and loadings

- Items 1 (group of node T1) and 15 (group of node T15) embedded,
  - Loading:
    1. Gravity according to  $-z$ ,
    2. Uniform rise in temperature of  $472.22 C^\circ$ ,
    3. Nodal forces.

Groups of node	T2	T3	T4, T10	T5, T9	T6, T7, T8	T11	T12, T13	T14
$Fz (N)$	-	-	-	-	-	-	- 117,720	- 176,580
	624,897	788,724	327,654	214,839	102,5145	222,687		

# Code\_Aster

Version  
default

Titre : SSSL101 - Tuyauterie : Problème de HOVGAARD  
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## 2 Reference solution

### 2.1 Method of calculating used for the reference solution

Reference solutions adopted to check modelings *Code\_Aster* are the following ones:

- for modelings *BEAM* : comparison with the codes: *POUX* , *ADL* and *TITUS-T* [1], using a modeling of type beam,
- for modelings *PIPE* : comparison with the code *ABAQUS* , using a modeling of type pipe. The number of mode of Fourier (  $M$  ) used during the calculation of the reference is identical to that used at the time as of calculations with *Code\_Aster*.
- for modelings beams, one also defines a moment equivalent to the point  $T1$  like  $M_{eq} = \sqrt{(MT^2 + MFY^2 + MFZ^2)}$  . The value of reference is that obtained with the hand with the results of modeling A (reference *AUTRE\_ASTER*).

### 2.2 Results of reference

Cas de Chargement	Displacement as in point 3 (group of node T3)	Modeling Beam ( <i>POUX</i> , <i>ADL</i> , <i>TITUS</i> )	Modeling Pipe: $M=3$ ( <i>ABAQUS</i> )	Modeling Pipe: $M=6$ ( <i>ABAQUS</i> )
Actual weight	$DX$	- 0.1658E-3	- 0.16517E-3	- 0.16512E-3
	$DY$	- 0.2040E-4	- 0.13870E-4	- 0.13946E-4
	$DZ$	- 0.8010E-5	- 0.80376E-5	- 0.80369E-5
Nodal force	$DX$	- 0.1651E-3	- 0.16445E-3	- 0.16441E-3
	$DY$	- 0.2080E-4	- 0.14245E-4	- 0.14320E-4
	$DZ$	- 0.9516E-5	- 0.10047E-4	- 0.10047E-4
Dilation	$DX$	- 6.1418E-3	- 6.3277E-3	- 6.3236E-3
	$DY$	- 13.090E-3	- 13.092E-3	- 13.093E-3
	$DZ$	16.799E-3	16.798E-3	16.798E-3

Case of loading	Moment are equivalent to the point $T1$	Reference (modeling A)	<i>AUTRE_ASTER</i>
Actual weight	$M_{eq}$	189.76886594440944	

### 2.3 Uncertainty on the solution

Uncertainty on the reference solution is fixed at 2% .

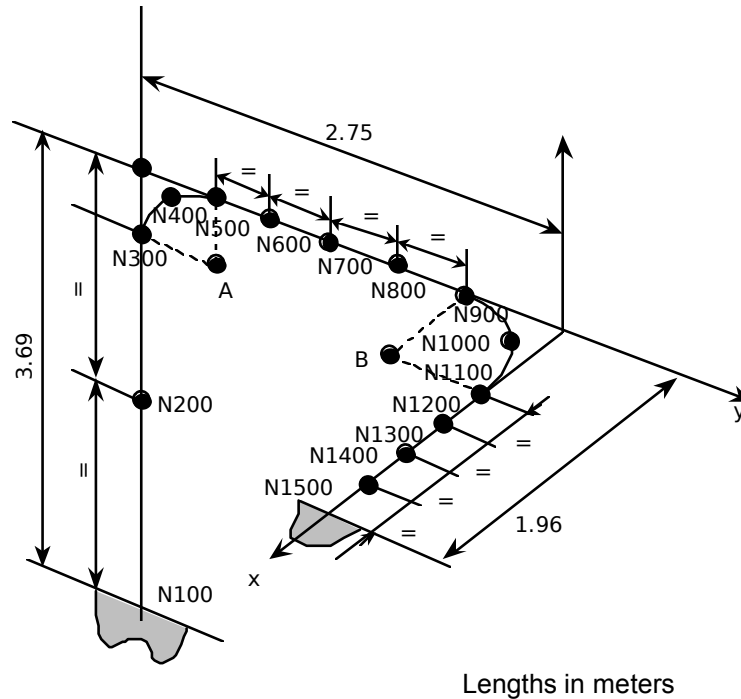
### 2.4 Bibliographical references

- [1] M.W. KELLOG Co. Design of Piping Systems. New York, 1956 - Problem n°5.9

## 3 Modeling A

### 3.1 Characteristics of modeling

The curved elements are modelled by right elements.  
A half curved element is modelled by 20 right elements.



### 3.2 Characteristics of the grid

Many nodes: 93  
Number of meshes and type: 92 POU\_D\_T

### 3.3 Sizes tested and results

Identification	Displacement	Type of reference	Value of reference	Precision (%)	
Actual weight	T 3	<i>DX</i>	'SOURCE_EXTERNE'	- 0.1658E-3	0.1
		<i>DY</i>	'SOURCE_EXTERNE'	- 0.2040E-4	0.1
		<i>DZ</i>	'SOURCE_EXTERNE'	- 0.8010E-5	0.1
Nodal force	T 3	<i>DX</i>	'SOURCE_EXTERNE'	- 0.1651E-3	0.1
		<i>DY</i>	'SOURCE_EXTERNE'	- 0.2080E-4	0.1
		<i>DZ</i>	'SOURCE_EXTERNE'	- 0.9516E-5	0.1
Dilation	T 3	<i>DX</i>	'SOURCE_EXTERNE'	- 6.1418E-3	0.1
		<i>DY</i>	'SOURCE_EXTERNE'	- 13.090E-3	0.1
		<i>DZ</i>	'SOURCE_EXTERNE'	16.799E-3	0.1

Identification	Type of reference	Value of reference	Precision (%)
Not T 1 - X 1 Field UT01_ELNO	'AUTRE_ASTER'	189.76886594440944	0.1

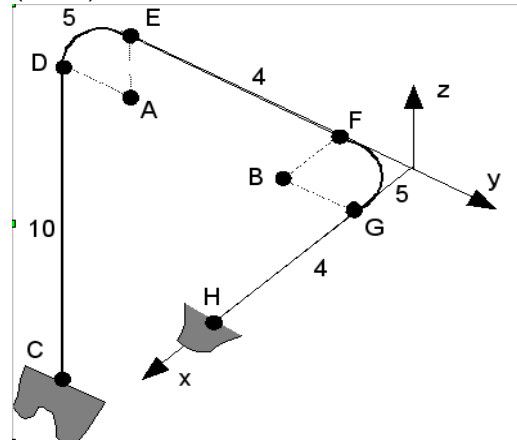
## 3.4 Notice

Differences between the results *Aster* and the reference solution beam are all lower than 0.05% .

## 4 Modeling C

### 4.1 Characteristics of modeling

Modeling PIPE (SEG3)



Lengths in meters

Boundary conditions: points *C* and *H*

- DDL beam:  $DX = DY = DZ = DRX = DRY = DRZ = 0$
- DDL hull:
  - $UIm = VIm = WIm = 0 (m=2,3)$
  - $UOm = VOm = WOm = 0 (m=2,3)$
  - $WI1 = WO1 = WO = 0$

### 4.2 Characteristics of the grid

Many nodes: 57  
Number of meshes and type: 28 SEG3

### 4.3 Values tested

Identification	Displacement	Reference pipe ( $M = 3$ )	%
	Not <i>D</i> <i>DX</i>	- 0.16517E-3	- 0.93
<b>Actual weight</b>	<i>DY</i>	- 0.13870E-4	- 9.80
	<i>DZ</i>	- 0.80376E-5	- 0.24
	Not <i>D</i> <i>DX</i>	- 0.16445E-3	- 0.94
<b>Nodal force</b>	<i>DY</i>	- 0.14245E-4	- 9.61
	<i>DZ</i>	- 0.10047E-4	- 0.20
	Not <i>D</i> <i>DX</i>	- 6.3277E-3	1.99
<b>Dilation</b>	<i>DY</i>	- 13.092E-3	0.08
	<i>DZ</i>	16.798E-3	- 0.93

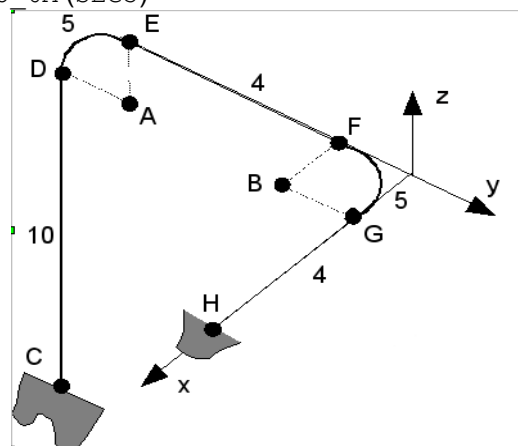
### 4.4 Notice

Results got with Code\_Aster are similar to those of ABAQUS by elements pipes except for displacement *DY* (actual weight and nodal force) where the variation is about 10%.

## 5 Modeling D

### 5.1 Characteristics of modeling

Modeling TUYAU\_6M (SEG3)



Lengths in meters

Boundary conditions: points *C* and *H*

- DDL beam:  $DX = DY = DZ = DRX = DRY = DRZ = 0$
- DDL hull:
  - $UIm = VIm = WIm = 0 (m = 2, 6)$
  - $UOm = VOm = WOm = 0 (m = 2, 6)$
  - $WI1 = WO1 = WO = 0$

### 5.2 Characteristics of the grid

Many nodes: 57  
Number of meshes and type: 28 SEG3

### 5.3 Sizes tested and results

Identification	Displacement	Reference pipe ( $M = 6$ )	%
Not <i>D</i>	<i>DX</i>	- 0.16512E-3	- 0.93
<b>Actual weight</b>	<i>DY</i>	- 0.13946E-4	- 9.78
	<i>DZ</i>	- 0.80369E-5	- 0.24
	Not <i>D</i>	<i>DX</i>	- 0.16441E-3
<b>Nodal force</b>	<i>DY</i>	- 0.14320E-4	- 9.58
	<i>DZ</i>	- 0.10047E-4	- 0.21
	Not <i>D</i>	<i>DX</i>	- 6.3236E-3
<b>Dilation</b>	<i>DY</i>	- 13.093E-3	0.08
	<i>DZ</i>	16.798E-3	0.49

### 5.4 Notice

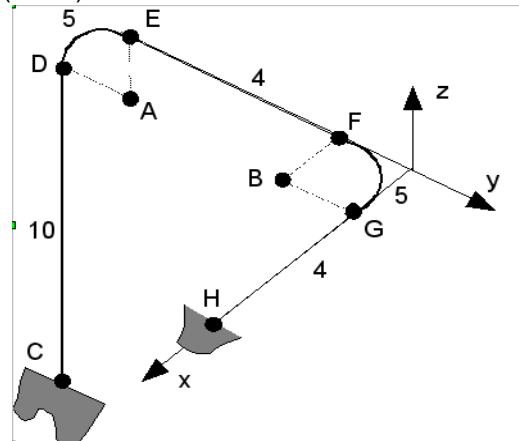
Results got with *Code\_Aster* are similar to those of ABAQUS for elements pipes except for displacement *DY* (actual weight and nodal force) where the variation is about 10%.



## 6 Modeling E

### 6.1 Characteristics of modeling

Modeling PIPE (SEG4)



Lengths in meters

Boundary conditions: points *C* and *H*

- DDL beam:  $DX = DY = DZ = DRX = DRY = DRZ = 0$
- DDL hull:
  - $UIm = VIm = WIm = 0 (m=2,3)$
  - $UOm = VOm = WOm = 0 (m=2,3)$
  - $WI1 = WO1 = WO = 0$

### 6.2 Characteristics of the grid

Many nodes: 85  
Number of meshes and type: 28 SEG4

### 6.3 Sizes tested and results

Identification	Displacement	Reference pipe ( $M=3$ )	%
<b>Actual weight</b>	Not <i>D</i> <i>DX</i>	-0.16517E-03	-0.22
	<i>DY</i>	-0.13870E-04	-4.73
	<i>DZ</i>	-0.80376E-05	-0.18
<b>Nodal force</b>	Not <i>D</i> <i>DX</i>	-0.16445E-03	-0.37
	<i>DY</i>	-0.14245E-04	-1.74
	<i>DZ</i>	-0.10047E-04	-0.50
<b>Dilation</b>	Not <i>D</i> <i>DX</i>	-6.3277E-03	0.02
	<i>DY</i>	-13.092E-03	0.10
	<i>DZ</i>	16.798E-03	0.27

### 6.4 Remarks

Grid in SEG4 is obtained starting from a grid SEG3 with the order CREA\_MAILLAGE, MODI\_MAILLE with the option 'SEG3\_4'. It is important that the node medium of SEG3 that is to say well in the medium, it Code\_Aster check this condition with a tolerance.

Results got with Code\_Aster are similar to those of ABAQUS with elements pipes except for displacement *DY* (actual weight and nodal force) or the variation is about 5% and 2%.

## 7 Summary of the results

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This CAS-test makes it possible to test a noncoplanar piping.

### Modeling beam:

The results are similar to the reference solution (modeling beam: average of results of 3 codes).

### Modeling pipe:

Results *Code\_Aster* are similar to those of ABAQUS (for elements pipes), except for displacement  $DY$  and for the loadings actual weight and nodal forces where the variation with the reference solution is more important with the meshes SEG3 ( 10% ) that with the meshes SEG4 ( 5% ).

The thermal loading of dilation gives similar results.